

## Document Cover Sheet

Dacro Industries Inc.

9325 - 51<sup>st</sup> Avenue Edmonton, Alberta T6E 4W8 ph: (780) 434-8900 fax: (780) 433-3138

	JOB NO.	PURCHASE ORDER NO.		
PO	085354A	P	3205	02
	EQUIP. NO.	VDR Code-Unique No.	Submission	
V.P.	8-V-851	C01-003	3	







<b>WorleyParsons</b> resources & energy REVIEW STAMP	
PROJECT NO. <u>085354A</u>	
1. REVIEWED & ACCEPTED	<input checked="" type="checkbox"/>
MANUFACTURE MAY PROCEED.	
2. REVIEWED & ACCEPTED AS MARKED, REVISE & RE-SUBMIT.	<input type="checkbox"/>
MANUFACTURE MAY PROCEED.	
3. REVISE AS MARKED AND RE-SUBMIT. MANUFACTURE SHALL NOT PROCEED.	<input type="checkbox"/>
4. REVIEW NOT REQUIRED. FOR INFORMATION ONLY.	<input type="checkbox"/>
MANUFACTURE MAY PROCEED.	
Acceptance in any of these categories in no way relieves the Contractor/Supplier of its responsibility for the due and proper performance of the works in acceptance with the Contract/Purchase Order.	
NAME: <u>Ali Raissian</u>	
SIGNED: <u>A. Ra</u>	
DATE: <u>23 / Aug / 2012</u>	

Dacro Job No. :	12-864
Customer :	WorleyParsons
End User:	MEG Energy Corp.
Project Title:	Christina Lake Regional Project, Phase II B
Customer P.O. No.	085354A-P-3205-02
Equipment:	Sour Gas Inlet Separator
Tag No.:	8-V-851
Document No. :	12-864-C-1
Description :	Pressure Boundary Calculations

Total Number of Pages: 21 including cover page

Date	Rev.	Description	Submitted By
2/27/2012	0	Original Issue	JW
5/10/2012	1	Revised As Noted	JW
7/16/2012	2	Revised Per Customer Comments	JW
7/16/2012	3	Updated Pressure & Test Summaries	JW

# Table Of Contents

- 1. Deficiencies Summary
-  2. Pressure Summary
-  3. Thickness Summary
-  4. Hydrostatic Test
-  5. Corroded Hydrostatic Test
- 6. Left Head
- 7. Straight Flange on Left Head
- 8. Straight Flange on Right Head
- 9. Shell #1/#2
- 10. Stiffening Ring

**Deficiencies Summary****Deficiencies for Platform #2**

Packed bed diameter (2,133 mm) is greater than inner diameter of Top of vessel (609.6 mm)

## Pressure Summary

## Pressure Summary for Chamber bounded by Right Head and Left Head

3

Identifier	P Design (kPa)	T Design (°C)	MAWP (kPa)	MAP (kPa)	MAEP (kPa)	T <sub>e</sub> external (°C)	MDMT (°C)	MDMT Exemption		Impact Tested
<u>Left Head</u>	862	150	866.16	1,305.66	160.2	150	-48	Note 1		No
<u>Straight Flange on Left Head</u>	862	150	1,192.12	1,631.66	180.89	150	-48	Note 2		No
<u>Shell #1/#2</u>	862	150	1,192.12	1,631.66	126.22	150	-48	Note 2		No
<u>Shell #2</u>	862	150	1,192.12	1,631.66	126.22	150	-48	Note 2		No
<u>Straight Flange on Right Head</u>	862	150	1,192.12	1,631.66	126.22	150	-48	Note 2		No
<u>Right Head</u>	862	150	866.16	1,305.66	160.2	150	-48	Note 3		No
<u>Stiffening Ring</u>	N/A	N/A	N/A	N/A	103	150	N/A	N/A		No
<u>Platform Support (J53, LC5)</u>	862	150	866.16	N/A	N/A	N/A	N/A	N/A		N/A
<u>Platform Support (J55, LC5)</u>	862	150	866.16	N/A	N/A	N/A	N/A	N/A		N/A
<u>Saddle</u>	862	150	866.16	N/A	N/A	N/A	N/A	N/A		N/A
<u>M1 24" CL150 RFWN (M1)</u>	862	150	866.16	1,305.66	126.22	150	-48	Nozzle	Note 4	No
								Pad	Note 5	No
<u>N1 16" CL150 RFWN (N1)</u>	862	150	866.16	1,305.66	160.19	150	-29	Nozzle	Note 6	No
								Pad	Note 7	No
<u>N10A 2" CL150 RFLWN (N10A/B)</u>	862	150	866.16	1,305.66	160.19	150	-48	Note 8		No
<u>N10B 2" CL150 RFLWN (N10B)</u>	862	150	866.16	1,305.66	126.22	150	-48	Note 8		No
<u>N11 3" CL150 RFWN (N11/N12)</u>	862	150	866.16	1,305.66	126.22	150	-48	Note 9		No
<u>N12 3" CL150 RFWN (N12)</u>	862	150	866.16	1,305.66	126.22	150	-48	Note 9		No
<u>N13 2" CL150 RFWN (N13/N16B)</u>	862	150	866.16	1,305.66	126.21	150	-48	Nozzle	Note 10	No
								Pad	Note 11	No
<u>N14 3" CL150 RFLWN (N14)</u>	862	150	866.16	1,305.66	126.22	150	-48	Note 12		No
<u>N15 2" CL150 RFWN (N15)</u>	862	150	866.16	1,305.66	126.21	150	-48	Note 9		No
<u>N16A 2" CL150 RFWN (N16A)</u>	862	150	866.16	1,305.66	126.21	150	-48	Note 9		No
<u>N16B 2" CL150 RFWN (N16B)</u>	862	150	866.16	1,305.66	126.21	150	-48	Nozzle	Note 10	No
								Pad	Note 11	No
<u>N2 12" CL150 RFWN (N2)</u>	862	150	866.16	1,305.66	126.21	150	-48	Nozzle	Note 13	No
								Pad	Note 14	No
<u>N3 4" CL150 RFLWN (N3)</u>	862	150	866.16	1,305.66	126.21	150	-48	Note 15		No
<u>N4 2" CL150 RFWN (N4)</u>	862	150	866.16	1,305.66	126.22	150	-48	Note 9		No
<u>N5A 3" CL150 RFWN (N5A/N6A)</u>	862	150	866.16	1,305.66	126.21	150	-48	Note 9		No
<u>N5B 3" CL150 RFWN (N5B/N6B)</u>	862	150	866.16	1,305.66	126.21	150	-48	Nozzle	Note 16	No
								Pad	Note 11	No
<u>N6A 3" CL150 RFWN (N6A)</u>	862	150	866.16	1,305.66	126.21	150	-48	Note 9		No
<u>N6B 3" CL150 RFWN (N6B)</u>	862	150	866.16	1,305.66	126.21	150	-48	Nozzle	Note 16	No
								Pad	Note 11	No
<u>N7 10" CL150 RFWN (N7/N8)</u>	862	150	866.16	1,305.66	126.21	150	-48	Nozzle	Note 17	No
								Pad	Note 14	No
<u>N8 10" CL150 RFWN (N8)</u>	862	150	866.16	1,305.66	126.21	150	-48	Nozzle	Note 17	No
								Pad	Note 14	No
<u>N9 3" CL600 RFWN (N9)</u>	862	150	866.16	1,305.66	126.22	150	-48	Note 18		No

Chamber design MDMT is -29 °C

Chamber rated MDMT is -29 °C @ 866.16 kPa

Chamber MAWP hot & corroded is 866.16 kPa @ 150 °C

Chamber MAP cold & new is 1,305.66 kPa @ 5 °C

Chamber MAEP is 103 kPa @ 150 °C

External pressure rating was governed by the vacuum ring Stiffening Ring.

#### Notes for MDMT Rating:

Note #	Exemption	Details
1.	<u>Straight Flange</u> governs MDMT	
2.	Material impact test exemption temperature from Fig UCS-66M Curve D = -48 °C 17 °C MDMT reduction per UCS-68(c) applies. Fig UCS-66.1M MDMT reduction = 15 °C, (coincident ratio = 0.7316) Rated MDMT of -80 °C is limited to -48 °C by UCS-66(b)(2)	UCS-66 governing thickness = 12.7 mm
3.	<u>Straight Flange</u> governs MDMT	
4.	Nozzle impact test exemption temperature from Fig UCS-66M Curve D = -48 °C 17 °C MDMT reduction per UCS-68(c) applies. Fig UCS-66.1M MDMT reduction = 15.5 °C, (coincident ratio = 0.7229) Rated MDMT of -80.5 °C is limited to -48 °C by UCS-66(b)(2)	UCS-66 governing thickness = 12.7 mm.
5.	Pad impact test exemption temperature from Fig UCS-66M Curve D = -48 °C 17 °C MDMT reduction per UCS-68(c) applies. Fig UCS-66.1M MDMT reduction = 15.5 °C, (coincident ratio = 0.7229) Rated MDMT of -80.5 °C is limited to -48 °C by UCS-66(b)(2)	UCS-66 governing thickness = 12.7 mm.
6.	Nozzle is impact test exempt per UG-20(f)	UCS-66 governing thickness = 12.7 mm.
7.	Pad impact test exemption temperature from Fig UCS-66M Curve D = -48 °C 17 °C MDMT reduction per UCS-68(c) applies. Fig UCS-66.1M MDMT reduction = 6.5 °C, (coincident ratio = 0.8844) Rated MDMT of -71.5 °C is limited to -48 °C by UCS-66(b)(2)	UCS-66 governing thickness = 12.7 mm.
8.	Nozzle impact test exemption temperature from Fig UCS-66M Curve C = -45 °C 17 °C MDMT reduction per UCS-68(c) applies. Fig UCS-66.1M MDMT reduction = 0.2 °C, (coincident ratio = 0.997) Rated MDMT of -62.2 °C is limited to -48 °C by UCS-66(b)(2)	UCS-66 governing thickness = 10.1 mm.
9.	Flange rating governs: Flange rated MDMT = -65 °C (UCS-68(c) applies.) Bolts rated MDMT per Fig UCS-66 note (c) = -48 °C	UCS-66(b)(1)(b)
10.	Nozzle is impact test exempt to -105 °C per UCS-66(b)(3) (coincident ratio = 0.0421).	
11.	Pad impact test exemption temperature from Fig UCS-66M Curve D = -48 °C 17 °C MDMT reduction per UCS-68(c) applies. Fig UCS-66.1M MDMT reduction = 15.1 °C, (coincident ratio = 0.7304) Rated MDMT of -80.1 °C is limited to -48 °C by UCS-66(b)(2)	UCS-66 governing thickness = 12.7 mm.
12.	Nozzle impact test exemption temperature from Fig UCS-66M Curve C = -37 °C 17 °C MDMT reduction per UCS-68(c) applies. Fig UCS-66.1M MDMT reduction = 15 °C, (coincident ratio = 0.7317) Rated MDMT of -69 °C is limited to -48 °C by UCS-66(b)(2)	UCS-66 governing thickness = 12.7 mm.
13.	Nozzle is impact test exempt to -105 °C per UCS-66(b)(3) (coincident ratio = 0.2523).	
14.	Pad impact test exemption temperature from Fig UCS-66M Curve D = -48 °C 17 °C MDMT reduction per UCS-68(c) applies. Fig UCS-66.1M MDMT reduction = 16 °C, (coincident ratio = 0.7143) Rated MDMT of -81 °C is limited to -48 °C by UCS-66(b)(2)	UCS-66 governing thickness = 12.7 mm.
15.	Nozzle impact test exemption temperature from Fig UCS-66M Curve B = -22 °C 17 °C MDMT reduction per UCS-68(c) applies. Fig UCS-66.1M MDMT reduction = 15 °C, (coincident ratio = 0.7317) Rated MDMT of -54 °C is limited to -48 °C by UCS-66(b)(2)	UCS-66 governing thickness = 12.7 mm.
16.	Nozzle is impact test exempt to -105 °C per UCS-66(b)(3) (coincident ratio = 0.0424).	
17.	Nozzle is impact test exempt to -105 °C per UCS-66(b)(3) (coincident ratio = 0.244).	
18.	Flange rating governs: Flange rated MDMT = -105 °C (UCS-68(c) applies.) Bolts rated MDMT per Fig UCS-66 note (c) = -48 °C	UCS-66(b)(3): Coincident ratio = 0.0848

Design notes are available on the Settings Summary page.

### Thickness Summary

Component Identifier	Material	Diameter (mm)	Length (mm)	Nominal t (mm)	Design t (mm)	Total Corrosion (mm)	Joint E	Load
<u>Left Head</u>	SA-516 70	2,133 ID	543.35	10.1*	10.07	3.2	1.00	Internal
<u>Straight Flange on Left Head</u>	SA-516 70	2,133 ID	50	12.7	10.82	3.2	1.00	External
<u>Shell #1/#2</u>	SA-516 70	2,133 ID	3,048	12.7	12.01	3.2	1.00	External
<u>Shell #2</u>	SA-516 70	2,133 ID	3,048	12.7	12.01	3.2	1.00	External
<u>Straight Flange on Right Head</u>	SA-516 70	2,133 ID	50	12.7	12.01	3.2	1.00	External
<u>Right Head</u>	SA-516 70	2,133 ID	543.35	10.1*	10.07	3.2	1.00	Internal

Nominal t: Vessel wall nominal thickness

Design t: Required vessel thickness due to governing loading + corrosion

Joint E: Longitudinal seam joint efficiency

\* Head minimum thickness after forming

#### Load

internal: Circumferential stress due to internal pressure governs

external: External pressure governs

Wind: Combined longitudinal stress of pressure + weight + wind governs

Seismic: Combined longitudinal stress of pressure + weight + seismic governs

### Hydrostatic Test

**Shop test pressure determination for Chamber bounded by Right Head and Left Head based on MAWP per UG-99(b)**

Shop hydrostatic test gauge pressure is 1,126.01 kPa at 5 °C (the chamber MAWP = 866.16 kPa)

The shop test is performed with the vessel in the horizontal position.

3

Identifier	Local test pressure kPa	Test liquid static head kPa	UG-99(b) stress ratio	UG-99(b) pressure factor	Stress during test MPa	Allowable test stress MPa	Stress excessive?
Left Head (1)	1,152.89	26.88	1	1.30	109.565	235.8	No
Straight Flange on Left Head	1,152.89	26.88	1	1.30	97.391	235.8	No
Shell #1/#2	1,152.89	26.88	1	1.30	97.391	235.8	No
Shell #2	1,152.89	26.88	1	1.30	97.391	235.8	No
Straight Flange on Right Head	1,152.89	26.88	1	1.30	97.391	235.8	No
Right Head	1,152.89	26.88	1	1.30	109.565	235.8	No
M1 24" CL150 RFWN (M1)	1,145.31	19.3	1	1.30	163.311	351	No
N1 16" CL150 RFWN (N1)	1,138.16	12.15	1	1.30	91.583	351	No
N10A 2" CL150 RFLWN (N10A/B)	1,150.53	24.52	1	1.30	118.762	351	No
N10B 2" CL150 RFLWN (N10B)	1,150.53	24.52	1	1.30	118.762	351	No
N11 3" CL150 RFWN (N11/N12)	1,131.91	5.9	1	1.30	112.729	351	No
N12 3" CL150 RFWN (N12)	1,131.91	5.9	1	1.30	112.729	351	No
N13 2" CL150 RFWN (N13/N16B)	1,151.67	25.66	1	1.30	48.356	351	No
N14 3" CL150 RFLWN (N14)	1,155.5	29.49	1	1.30	107.271	351	No
N15 2" CL150 RFWN (N15)	1,131.87	5.86	1	1.30	110.586	351	No
N16A 2" CL150 RFWN (N16A)	1,134.03	8.02	1	1.30	105.924	351	No
N16B 2" CL150 RFWN (N16B)	1,151.67	25.66	1	1.30	48.356	351	No
N2 12" CL150 RFWN (N2)	1,131.87	5.86	1	1.30	108.75	351	No
N3 4" CL150 RFLWN (N3)	1,155.5	29.49	1	1.30	105.743	351	No
N4 2" CL150 RFWN (N4)	1,131.91	5.9	1	1.30	105.726	351	No
N5A 3" CL150 RFWN (N5A/N6A)	1,132.28	6.27	1	1.30	112.766	351	No
N5B 3" CL150 RFWN (N5B/N6B)	1,151.78	25.77	1	1.30	53.154	351	No
N6A 3" CL150 RFWN (N6A)	1,132.28	6.27	1	1.30	112.766	351	No
N6B 3" CL150 RFWN (N6B)	1,151.78	25.77	1	1.30	53.154	351	No
N7 10" CL150 RFWN (N7/N8)	1,131.87	5.86	1	1.30	85.861	351	No
N8 10" CL150 RFWN (N8)	1,131.87	5.86	1	1.30	85.861	351	No
N9 3" CL600 RFWN (N9)	1,131.91	5.9	1	1.30	112.729	351	No

**Notes:**

- (1) Left Head limits the UG-99(b) stress ratio.
- (2)  $P_L$  stresses at nozzle openings have been estimated using the method described in PVP-Vol. 399, pages 77-82.
- (3)  $1.5 \times 0.9 \times S_y$  used as the basis for the maximum local primary membrane stress at the nozzle intersection  $P_L$ .
- (4) The zero degree angular position is assumed to be up, and the test liquid height is assumed to the top-most flange.

The field test condition has not been investigated for the Chamber bounded by Right Head and Left Head.

provision of UG-99(h) has been met.



## Corroded Hydrostatic Test

The shop test condition has not been investigated for the Chamber bounded by Right Head and Left Head.

**Field test pressure determination for Chamber bounded by Right Head and Left Head based on MAWP per UG-99(b)**

Field hydrostatic test gauge pressure is 1,126.01 kPa at 5 °C (the chamber MAWP = 866.16 kPa)

The field test is performed with the vessel in the horizontal position.

3

Identifier	Local test pressure kPa	Test liquid static head kPa	UG-99(b) stress ratio	UG-99(b) pressure factor	Stress during test MPa	Allowable test stress MPa	Stress excessive?
Left Head (1)	1,152.92	26.91	1	1.30	160.872	235.8	No
Straight Flange on Left Head	1,152.92	26.91	1	1.30	130.4	235.8	No
Shell #1/#2	1,152.92	26.91	1	1.30	130.4	235.8	No
Shell #2	1,152.92	26.91	1	1.30	130.4	235.8	No
Straight Flange on Right Head	1,152.92	26.91	1	1.30	130.4	235.8	No
Right Head	1,152.92	26.91	1	1.30	160.872	235.8	No
M1 24" CL150 RFWN (M1)	1,145.34	19.33	1	1.30	234.063	351	No
N1 16" CL150 RFWN (N1)	1,138.19	12.18	1	1.30	107.124	351	No
N10A 2" CL150 RFLWN (N10A/B)	1,150.56	24.55	1	1.30	183.468	351	No
N10B 2" CL150 RFLWN (N10B)	1,150.56	24.55	1	1.30	183.468	351	No
N11 3" CL150 RFWN (N11/N12)	1,131.91	5.9	1	1.30	168.085	351	No
N12 3" CL150 RFWN (N12)	1,131.91	5.9	1	1.30	168.085	351	No
N13 2" CL150 RFWN (N13/N16B)	1,151.7	25.69	1	1.30	55.495	351	No
N14 3" CL150 RFLWN (N14)	1,155.5	29.49	1	1.30	143.576	351	No
N15 2" CL150 RFWN (N15)	1,131.87	5.86	1	1.30	158.202	351	No
N16A 2" CL150 RFWN (N16A)	1,134.06	8.05	1	1.30	158.509	351	No
N16B 2" CL150 RFWN (N16B)	1,151.7	25.69	1	1.30	55.495	351	No
N2 12" CL150 RFWN (N2)	1,131.87	5.86	1	1.30	122.073	351	No
N3 4" CL150 RFLWN (N3)	1,155.5	29.49	1	1.30	136.868	351	No
N4 2" CL150 RFWN (N4)	1,131.91	5.9	1	1.30	158.208	351	No
N5A 3" CL150 RFWN (N5A/N6A)	1,132.28	6.27	1	1.30	168.14	351	No
N5B 3" CL150 RFWN (N5B/N6B)	1,151.81	25.8	1	1.30	55.501	351	No
N6A 3" CL150 RFWN (N6A)	1,132.28	6.27	1	1.30	168.14	351	No
N6B 3" CL150 RFWN (N6B)	1,151.81	25.8	1	1.30	55.501	351	No
N7 10" CL150 RFWN (N7/N8)	1,131.87	5.86	1	1.30	88.053	351	No
N8 10" CL150 RFWN (N8)	1,131.87	5.86	1	1.30	88.053	351	No
N9 3" CL600 RFWN (N9)	1,131.91	5.9	1	1.30	168.085	351	No

**Notes:**

- (1) Left Head limits the UG-99(b) stress ratio.
- (2)  $P_L$  stresses at nozzle openings have been estimated using the method described in PVP-Vol. 399, pages 77-82.
- (3)  $1.5 \cdot 0.9 \cdot S_y$  used as the basis for the maximum local primary membrane stress at the nozzle intersection  $P_L$ .
- (4) The zero degree angular position is assumed to be up, and the test liquid height is assumed to the top-most flange.

The test temperature of 5 °C is warmer than the minimum recommended temperature of -12 °C so the brittle fracture provision of UG-99(h) has been met.

## Left Head

## ASME Section VIII, Division 1, 2010 Edition Metric

Component: Ellipsoidal Head  
 Material Specification: SA-516 70 (II-D Metric p.18, ln. 19)  
Straight Flange governs MDMT

Internal design pressure:  $P = 862 \text{ kPa @ } 150^\circ\text{C}$   
 External design pressure:  $P_e = 103 \text{ kPa @ } 150^\circ\text{C}$

## Static liquid head:

$P_s = 26.91 \text{ kPa}$  (SG=1,  $H_s=2746.7 \text{ mm}$  Operating head)  
 $P_{th} = 26.88 \text{ kPa}$  (SG=1,  $H_s=2743.5 \text{ mm}$  Horizontal test head)

Corrosion allowance: Inner C = 3.2 mm Outer C = 0 mm

Design MDMT =  $-29^\circ\text{C}$  No impact test performed  
 Rated MDMT =  $-48^\circ\text{C}$  Material is normalized  
 Material is produced to fine grain practice  
 PWHT is performed  
 Do not Optimize MDMT / Find MAWP

Radiography: Category A joints - Seamless No RT  
 Head to shell seam - Full UW-11(a) Type 1

Estimated weight\*: new = 442.4 kg corr = 305.5 kg  
 Capacity\*: new = 1,449 liters corr = 1,465.4 liters  
 \* includes straight flange

Inner diameter = 2133 mm  
 Minimum head thickness = 10.1 mm  
 Head ratio  $D/2h$  = 2 (new)  
 Head ratio  $D/2h$  = 1.994 (corroded)  
 Straight flange length  $L_{sf}$  = 50 mm  
 Nominal straight flange thickness  $t_{sf}$  = 12.7 mm

Insulation thk\*: 50 mm density:  $240.277 \text{ kg/m}^3$  weight: 66.2151 kg  
 Insulation support ring spacing: 0 mm individual weight: 0 kg total weight: 0 kg  
 \* includes straight flange if applicable

## Results Summary

The governing condition is internal pressure.

Minimum thickness per UG-16 =  $1.5 \text{ mm} + 3.2 \text{ mm} = 4.7 \text{ mm}$   
 Design thickness due to internal pressure (t) = 10.07 mm  
 Design thickness due to external pressure ( $t_e$ ) = 8.74 mm  
 Maximum allowable working pressure (MAWP) = 866.16 kPa  
 Maximum allowable pressure (MAP) = 1,305.66 kPa  
 Maximum allowable external pressure (MAEP) = 160.2 kPa

## K (Corroded)

$$K = (1/6) * [2 + (D / (2 * h))]^2 = (1/6) * [2 + (2,139.4 / (2 * 536.45))]^2 = 0.996029$$

**K (New)**

$$K = (1/6) * [2 + (D / (2 * h))^2] = (1/6) * [2 + (2,133 / (2 * 533.25))^2] = 1$$

**Design thickness for internal pressure, (Corroded at 150 °C) Appendix 1-4(c)**

$$\begin{aligned} t &= P * D * K / (2 * S * E - 0.2 * P) + \text{Corrosion} \\ &= 888.91 * 2,139.4 * 0.996029 / (2 * 138,000.002 * 1 - 0.2 * 888.91) + 3.2 \\ &= 10.07 \text{ mm} \end{aligned}$$

The head internal pressure design thickness is 10.07 mm.

**Maximum allowable working pressure, (Corroded at 150 °C) Appendix 1-4(c)**

$$\begin{aligned} P &= 2 * S * E * t / (K * D + 0.2 * t) - P_s \\ &= 2 * 138,000.002 * 1 * 6.9 / (0.996029 * 2,139.4 + 0.2 * 6.9) - 26.91 \\ &= 866.16 \text{ kPa} \end{aligned}$$

The maximum allowable working pressure (MAWP) is 866.16 kPa.

**Maximum allowable pressure, (New at 5 °C) Appendix 1-4(c)**

$$\begin{aligned} P &= 2 * S * E * t / (K * D + 0.2 * t) - P_s \\ &= 2 * 138,000.002 * 1 * 10.1 / (1 * 2,133 + 0.2 * 10.1) - 0 \\ &= 1,305.66 \text{ kPa} \end{aligned}$$

The maximum allowable pressure (MAP) is 1,305.66 kPa.

**Design thickness for external pressure, (Corroded at 150 °C) UG-33(d)**

Equivalent outside spherical radius ( $R_o$ )

$$\begin{aligned} R_o &= K_o * D_o \\ &= 0.8916 * 2,153.2 \\ &= 1,919.87 \text{ mm} \end{aligned}$$

$$\begin{aligned} A &= 0.125 / (R_o / t) \\ &= 0.125 / (1,919.87 / 5.54) \\ &= 0.00036 \end{aligned}$$

From Table CS-2       $B = 35.7247$   
Metric:                       $B = \text{MPa}$

$$\begin{aligned} P_a &= B / (R_o / t) \\ &= 35,724.67 / (1,919.87 / 5.54) \\ &= 103 \text{ kPa} \end{aligned}$$

$$t = 5.54 \text{ mm} + \text{Corrosion} = 5.54 \text{ mm} + 3.2 \text{ mm} = 8.74 \text{ mm}$$

Check the external pressure per UG-33(a)(1) Appendix 1-4(c)

$$\begin{aligned} t &= 1.67 * P_e * D * K / (2 * S * E - 0.2 * 1.67 * P_e) + \text{Corrosion} \\ &= 1.67 * 103 * 2,139.4 * 0.996029 / (2 * 138,000.002 * 1 - 0.2 * 1.67 * 103) + 3.2 \\ &= 4.53 \text{ mm} \end{aligned}$$

The head external pressure design thickness ( $t_e$ ) is 8.74 mm.

**Maximum Allowable External Pressure, (Corroded at 150 °C) UG-33(d)**

Equivalent outside spherical radius ( $R_o$ )

$$R_o = K_o * D_o$$

$$= 0.8916 \times 2,153.2$$

$$= 1,919.87 \text{ mm}$$

$$A = 0.125 / (R_o / t)$$

$$= 0.125 / (1,919.87 / 6.9)$$

$$= 0.000449$$

From Table CS-2       $B = 44.5761$   
Metric:                      MPa

$$P_a = B / (R_o / t)$$

$$= 44,576.09 / (1,919.87 / 6.9)$$

$$= 160.197 \text{ kPa}$$

**Check the Maximum External Pressure, UG-33(a)(1) Appendix 1-4(c)**

$$P = 2 \cdot S \cdot E \cdot t / ((K \cdot D + 0.2 \cdot t) \cdot 1.67) - P_{s2}$$

$$= 2 \cdot 138,000.002 \cdot 1 \cdot 6.9 / ((0.996029 \cdot 2,139.4 + 0.2 \cdot 6.9) \cdot 1.67) - 0$$

$$= 534.78 \text{ kPa}$$

The maximum allowable external pressure (MAEP) is 160.2 kPa.

**% Extreme fiber elongation - UCS-79(d)**

$$EFE = (75 \cdot t / R_i) \cdot (1 - R_i / R_o)$$

$$= (75 \cdot 12.7 / 368.96) \cdot (1 - 368.96 / \infty)$$

$$= 2.5816\%$$

The extreme fiber elongation does not exceed 5%.

## Straight Flange on Left Head

## ASME Section VIII Division 1, 2010 Edition Metric

Component: Straight Flange  
 Material specification: SA-516 70 (II-D Metric p. 18, In. 19)  
 Material impact test exemption temperature from Fig UCS-66M Curve D = -48 °C  
 17 °C MDMT reduction per UCS-68(c) applies.  
 Fig UCS-66.1M MDMT reduction = 15 °C, (coincident ratio = 0.7316)  
 Rated MDMT of -80°C is limited to -48°C by UCS-66(b)(2)  
 UCS-66 governing thickness = 12.7 mm

Internal design pressure:  $P = 862 \text{ kPa @ } 150 \text{ °C}$   
 External design pressure:  $P_e = 103 \text{ kPa @ } 150 \text{ °C}$

## Static liquid head:

$P_s = 26.91 \text{ kPa}$  (SG = 1,  $H_s = 2,746.7 \text{ mm}$ , Operating head)  
 $P_{th} = 26.88 \text{ kPa}$  (SG = 1,  $H_s = 2,743.5 \text{ mm}$ , Horizontal test head)

Corrosion allowance      Inner C = 3.2 mm      Outer C = 0 mm

Design MDMT = -29 °C      No impact test performed  
 Rated MDMT = -48 °C      Material is normalized  
    Material is produced to Fine Grain Practice  
    PWHT is performed

Radiography:      Longitudinal joint -      Seamless No RT  
                                  Circumferential joint -      Full UW-11(a) Type 1

Estimated weight New = 33.5 kg      corr = 25.1 kg  
 Capacity      New = 178.67 liters      corr = 179.74 liters

ID = 2,133 mm  
 Length = 50 mm  
 $L_c$   
 $t = 12.7 \text{ mm}$   
 Insulation thk: 50 mm      density: 240.28 kg/m<sup>3</sup>      Weight: 0 kg

## Design thickness, (at 150 °C) UG-27(c)(1)

$$t = P \cdot R / (S \cdot E - 0.60 \cdot P) + \text{Corrosion}$$

$$= 888.91 \cdot 1,069.7 / (138,000 \cdot 1.00 - 0.60 \cdot 888.91) + 3.2$$

$$= 10.12 \text{ mm}$$

## Maximum allowable working pressure, (at 150 °C) UG-27(c)(1)

$$P = S \cdot E \cdot t / (R + 0.60 \cdot t) - P_s$$

$$= 138,000 \cdot 1.00 \cdot 9.5 / (1,069.7 + 0.60 \cdot 9.5) - 26.91$$

$$= 1,192.12 \text{ kPa}$$

## Maximum allowable pressure, (at 5 °C) UG-27(c)(1)

$$P = S \cdot E \cdot t / (R + 0.60 \cdot t)$$

$$= 138,000 \cdot 1.00 \cdot 12.7 / (1,066.5 + 0.60 \cdot 12.7)$$

$$= 1,631.66 \text{ kPa}$$

**External Pressure, (Corroded & at 150 °C) UG-28(c)**

$$\begin{aligned}
 L / D_o &= 2,719.82 / 2,158.4 = 1.2601 \\
 D_o / t &= 2,158.4 / 7.62 = 283.2441 \\
 \text{From table G: } A &= 0.000221 \\
 \text{From table CS-2 } B &= 21.8805 \text{ MPa} \\
 \text{Metric:}
 \end{aligned}$$

$$\begin{aligned}
 P_a &= 4*B / (3*(D_o / t)) \\
 &= 4*21,880.55 / (3*(2,158.4 / 7.62)) \\
 &= 103 \text{ kPa}
 \end{aligned}$$

**Design thickness for external pressure  $P_a = 103 \text{ kPa}$** 

$$t_a = t + \text{Corrosion} = 7.62 + 3.2 = 10.82 \text{ mm}$$

**Maximum Allowable External Pressure, (Corroded & at 150 °C) UG-28(c)**

$$\begin{aligned}
 L / D_o &= 2,719.82 / 2,158.4 = 1.2601 \\
 D_o / t &= 2,158.4 / 9.5 = 227.2096 \\
 \text{From table G: } A &= 0.000311 \\
 \text{From table CS-2 } B &= 30.8252 \text{ MPa} \\
 \text{Metric:}
 \end{aligned}$$

$$\begin{aligned}
 P_a &= 4*B / (3*(D_o / t)) \\
 &= 4*30,825.17 / (3*(2,158.4 / 9.5)) \\
 &= 180.89 \text{ kPa}
 \end{aligned}$$

**% Extreme fiber elongation - UCS-79(d)**

$$\begin{aligned}
 \text{EFE} &= (50*t / R_t) * (1 - R_t / R_o) \\
 &= (50*12.7 / 1,072.85) * (1 - 1,072.85 / \infty) \\
 &= 0.5919\%
 \end{aligned}$$

The extreme fiber elongation does not exceed 5%.

## Straight Flange on Right Head

### ASME Section VIII Division 1, 2010 Edition Metric

Component: Straight Flange  
 Material specification: SA-516 70 (II-D Metric p. 18, ln. 19)  
 Material impact test exemption temperature from Fig UCS-66M Curve D = -48 °C  
 17 °C MDMT reduction per UCS-68(c) applies.  
 Fig UCS-66.1M MDMT reduction = 15 °C, (coincident ratio = 0.7316)  
 Rated MDMT of -80°C is limited to -48°C by UCS-66(b)(2)  
 UCS-66 governing thickness = 12.7 mm

Internal design pressure:  $P = 862 \text{ kPa @ } 150 \text{ °C}$   
 External design pressure:  $P_e = 103 \text{ kPa @ } 150 \text{ °C}$

#### Static liquid head:

$P_s = 26.91 \text{ kPa}$  (SG = 1,  $H_s = 2,746.7 \text{ mm}$ , Operating head)  
 $P_{th} = 26.88 \text{ kPa}$  (SG = 1,  $H_s = 2,743.5 \text{ mm}$ , Horizontal test head)

Corrosion allowance      Inner C = 3.2 mm      Outer C = 0 mm

Design MDMT = -29 °C      No impact test performed  
 Rated MDMT = -48 °C      Material is normalized  
    Material is produced to Fine Grain Practice  
    PWHT is performed

Radiography:      Longitudinal joint -      Seamless No RT  
                                  Circumferential joint -      Full UW-11(a) Type 1

Estimated weight New = 33.5 kg      corr = 25.1 kg  
 Capacity      New = 178.67 liters      corr = 179.74 liters

ID = 2,133 mm  
 Length = 50 mm  
 $L_c$   
 $t = 12.7 \text{ mm}$

#### Design thickness, (at 150 °C) UG-27(c)(1)

$t = P \cdot R / (S \cdot E - 0.60 \cdot P) + \text{Corrosion}$   
 $= 888.91 \cdot 1,069.7 / (138,000 \cdot 1.00 - 0.60 \cdot 888.91) + 3.2$   
 $= 10.12 \text{ mm}$

#### Maximum allowable working pressure, (at 150 °C) UG-27(c)(1)

$P = S \cdot E \cdot t / (R + 0.60 \cdot t) - P_s$   
 $= 138,000 \cdot 1.00 \cdot 9.5 / (1,069.7 + 0.60 \cdot 9.5) - 26.91$   
 $= 1,192.12 \text{ kPa}$

#### Maximum allowable pressure, (at 5 °C) UG-27(c)(1)

$P = S \cdot E \cdot t / (R + 0.60 \cdot t)$   
 $= 138,000 \cdot 1.00 \cdot 12.7 / (1,066.5 + 0.60 \cdot 12.7)$   
 $= 1,631.66 \text{ kPa}$



**External Pressure, (Corroded & at 150 °C) UG-28(c)**

$$\begin{aligned}
 L / D_o &= 3,833.82 / 2,158.4 = 1.7762 \\
 D_o / t &= 2,158.4 / 8.81 = 245.0701 \\
 \text{From table G: } A &= 0.000192 \\
 \text{From table CS-2 } B &= 18.9316 \text{ MPa} \\
 \text{Metric:}
 \end{aligned}$$

$$\begin{aligned}
 P_a &= 4*B / (3*(D_o / t)) \\
 &= 4*18,931.61 / (3*(2,158.4 / 8.81)) \\
 &= 103 \text{ kPa}
 \end{aligned}$$

**Design thickness for external pressure  $P_a = 103 \text{ kPa}$** 

$$t_a = t + \text{Corrosion} = 8.81 + 3.2 = 12.01 \text{ mm}$$

**Maximum Allowable External Pressure, (Corroded & at 150 °C) UG-28(c)**

$$\begin{aligned}
 L / D_o &= 3,833.82 / 2,158.4 = 1.7762 \\
 D_o / t &= 2,158.4 / 9.5 = 227.2096 \\
 \text{From table G: } A &= 0.000217 \\
 \text{From table CS-2 } B &= 21.5086 \text{ MPa} \\
 \text{Metric:}
 \end{aligned}$$

$$\begin{aligned}
 P_a &= 4*B / (3*(D_o / t)) \\
 &= 4*21,508.62 / (3*(2,158.4 / 9.5)) \\
 &= 126.22 \text{ kPa}
 \end{aligned}$$

**% Extreme fiber elongation - UCS-79(d)**

$$\begin{aligned}
 \text{EFE} &= (50*t / R_t) * (1 - R_t / R_o) \\
 &= (50*12.7 / 1,072.85) * (1 - 1,072.85 / \infty) \\
 &= 0.5919\%
 \end{aligned}$$

The extreme fiber elongation does not exceed 5%.

## Shell #1/#2

## ASME Section VIII Division 1, 2010 Edition Metric

Component: Cylinder  
 Material specification: SA-516 70 (II-D Metric p. 18, In. 19)  
 Material impact test exemption temperature from Fig UCS-66M Curve D = -48 °C  
 17 °C MDMT reduction per UCS-68(c) applies.  
 Fig UCS-66.1M MDMT reduction = 15 °C, (coincident ratio = 0.7316)  
 Rated MDMT of -80 °C is limited to -48 °C by UCS-66(b)(2)  
 UCS-66 governing thickness = 12.7 mm

Internal design pressure:  $P = 862 \text{ kPa @ } 150 \text{ °C}$   
 External design pressure:  $P_e = 103 \text{ kPa @ } 150 \text{ °C}$

## Static liquid head:

$P_s = 26.91 \text{ kPa}$  (SG = 1,  $H_s = 2,746.7 \text{ mm}$ , Operating head)  
 $P_{th} = 26.88 \text{ kPa}$  (SG = 1,  $H_s = 2,743.5 \text{ mm}$ , Horizontal test head)

Corrosion allowance Inner C = 3.2 mm Outer C = 0 mm

Design MDMT = -29 °C No impact test performed  
 Rated MDMT = -48 °C Material is normalized  
 Material is produced to Fine Grain Practice  
 PWHT is performed

Radiography: Longitudinal joint - Full UW-11(a) Type 1  
 Left circumferential joint - Full UW-11(a) Type 1  
 Right circumferential joint - Full UW-11(a) Type 1

Estimated weight New = 2,011.8 kg corr = 1,507.1 kg  
 Capacity New = 10,891.47 liters corr = 10,956.94 liters

ID = 2,133 mm  
 Length = 3,048 mm  
 $L_c = 12.7 \text{ mm}$   
 $t = 12.7 \text{ mm}$   
 Insulation thk: 50 mm density: 240.28 kg/m<sup>3</sup> Weight: 254.1 kg

## Design thickness, (at 150 °C) UG-27(c)(1)

$t = P \cdot R / (S \cdot E - 0.60 \cdot P) + \text{Corrosion}$   
 $= 888.91 \cdot 1,069.7 / (138,000 \cdot 1.00 - 0.60 \cdot 888.91) + 3.2$   
 $= 10.12 \text{ mm}$

## Maximum allowable working pressure, (at 150 °C) UG-27(c)(1)

$P = S \cdot E \cdot t / (R + 0.60 \cdot t) - P_s$   
 $= 138,000 \cdot 1.00 \cdot 9.5 / (1,069.7 + 0.60 \cdot 9.5) - 26.91$   
 $= 1,192.12 \text{ kPa}$

## Maximum allowable pressure, (at 5 °C) UG-27(c)(1)

$P = S \cdot E \cdot t / (R + 0.60 \cdot t)$   
 $= 138,000 \cdot 1.00 \cdot 12.7 / (1,066.5 + 0.60 \cdot 12.7)$   
 $= 1,631.66 \text{ kPa}$

**External Pressure, (Corroded & at 150 °C) UG-28(c)**

$$\begin{aligned}
 L / D_o &= 3,833.82 / 2,158.4 = 1.7762 \\
 D_o / t &= 2,158.4 / 8.81 = 245.0701 \\
 \text{From table G:} \quad A &= 0.000192 \\
 \text{From table CS-2} \quad B &= 18.9316 \text{ MPa} \\
 \text{Metric:}
 \end{aligned}$$

$$\begin{aligned}
 P_a &= 4*B / (3*(D_o / t)) \\
 &= 4*18,931.61 / (3*(2,158.4 / 8.81)) \\
 &= 103 \text{ kPa}
 \end{aligned}$$

**Design thickness for external pressure  $P_a = 103 \text{ kPa}$** 

$$t_a = t + \text{Corrosion} = 8.81 + 3.2 = 12.01 \text{ mm}$$

**Maximum Allowable External Pressure, (Corroded & at 150 °C) UG-28(c)**

$$\begin{aligned}
 L / D_o &= 3,833.82 / 2,158.4 = 1.7762 \\
 D_o / t &= 2,158.4 / 9.5 = 227.2096 \\
 \text{From table G:} \quad A &= 0.000217 \\
 \text{From table CS-2} \quad B &= 21.5086 \text{ MPa} \\
 \text{Metric:}
 \end{aligned}$$

$$\begin{aligned}
 P_a &= 4*B / (3*(D_o / t)) \\
 &= 4*21,508.62 / (3*(2,158.4 / 9.5)) \\
 &= 126.22 \text{ kPa}
 \end{aligned}$$

**% Extreme fiber elongation - UCS-79(d)**

$$\begin{aligned}
 \text{EFE} &= (50*t / R_f) * (1 - R_f / R_o) \\
 &= (50*12.7 / 1,072.85) * (1 - 1,072.85 / \infty) \\
 &= 0.5919\%
 \end{aligned}$$

The extreme fiber elongation does not exceed 5%.

## Stiffening Ring

## Stiffener ring calculations per UG-29(a)

Ring type:	Flat bar	
Ring description:	3/8x4 Flat Bar	
Ring material:	SA-516 70 (II-D Metric p. 18, In. 19)	
External pressure:	103	kPa
Ring is located:	outside the vessel	
Distance from ring neutral axis to datum:	3,605	mm
Ring corrosion allowance:	0	mm
Distance to previous support:	2,719.82	mm
Distance to next support:	3,833.82	mm

$$L / D_o = 3,833.82 / 2,158.4 = 1.7762$$

$$D_o / t = 2,158.4 / 8.81 = 245.0695$$

$$\text{From Table G: } A = 0.000192$$

$$\text{From Table CS-2 Metric: } B = 18.932 \text{ MPa}$$

$$\begin{aligned} P_a &= 4*B / (3*(D_o / t)) \\ &= 4*18.932 / (3*(2,158.4 / 8.81)) \\ &= 0.103 \text{ MPa} \end{aligned}$$

$$\begin{aligned} B &= 0.75*P*D_o / (t + A_s / L_s) \\ &= 0.75*103*2,158.4 / (8.81 + 967.74 / 3,276.82) / 1000 \\ &= 18.32 \text{ MPa} \end{aligned}$$

$$\text{From Table CS-2 Metric: } A = 0.00018539 \text{ (ring, } 150^\circ\text{C)}$$

$$\begin{aligned} I_s' &= [D_o^2*L_s*(t + A_s / L_s)*A] / 10.9 \\ &= [2,158.4^2*3,276.82*(8.81 + 967.74 / 3,276.82)*0.00018539] / 10.9 / 10000 \\ &= 236.31 \text{ cm}^4 \end{aligned}$$

$I'$  for the composite corroded shell-ring cross section is 265.71 cm<sup>4</sup>

As  $I' \geq I_s'$  a 3/8x4 Flat Bar stiffener is adequate for an external pressure of 103 kPa.

## Check the stiffener ring attachment welds per UG-30

Fillet weld is:	Continuous both sides
Fillet weld leg size:	6 mm
Vessel thickness at weld location, new:	12.7 mm
Vessel corrosion allowance at weld location:	3.2 mm
Stiffener thickness at weld location:	9.53 mm

Per UG-30(f)(1) the minimum attachment weld size is 6 mm

The fillet weld size of 6 mm is adequate per UG-30(f)(1).

$$\text{Radial pressure load, } P*L_s = 103*3,276.82 / 100 = 3,375.12 \text{ N/cm}$$

$$\text{Radial shear load, } V = 0.01*P*L_s*D_o = 0.01*103 / 1000*3,276.82*2,158.4 = 7,284.86 \text{ N}$$

$$\text{First moment of area, } Q = 14.96*2.18 = 32.6446 \text{ cm}^3$$

Weld shear flow,  $q = V*Q / I' = 894.9979 \text{ N/cm}$

Combined weld load,  $f_w = \text{Sqr}(3,375.1208^2 + 894.9979^2) = 3,491.77 \text{ N/cm}$

Allowable weld stress per UW-18(d)  $S_w = 0.55*S = 0.55*138 = 75.9 \text{ MPa}$

**Fillet weld size required to resist radial pressure and shear**

$$\begin{aligned}
 &= f_w * (d_{\text{weld segment}} + d_{\text{toe}}) / (S_w * d_{\text{weld total}}) + \text{corrosion} \\
 &= 3,491.77 * (25.4 + 0) / (10 * 75.9 * 50.8) + 0 \\
 &= 2.3 \text{ mm}
 \end{aligned}$$

The fillet weld size of 6 mm is adequate to resist radial pressure and shear.